

## MODELING BOUNDARY LAYER FLOW AND HEAT TRANSFER OF A PARTICULATE SUSPENSION

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**ABSTRACT.** The flow of a steady viscous incompressible fluid with uniformly distributed suspended particles past a thin heated semi-infinite flat plate is considered. Finite difference technique with non-uniform grid is used to investigate the effect of various flow parameters on the flow and heat transfer characteristics of the particulate suspension. Heat is diffused away from the heated surface for smaller values of Prandtl number  $Pr$  more rapidly than that of higher values of  $Pr$ . The magnitude of particle velocity and particle phase density are increased whereas the particle temperature is reduced due to the presence of coarser particles with high material density. Higher values of Prandtl number  $Pr$  is to increase the magnitude of carrier fluid velocity, particle velocity, particle phase density and Nusselt number.

**Key Words:** Volume fraction, Suspended particulate matter, Particulate Suspension, Slip velocity, Heat Transfer.

**AMS (MOS) Subject Classification.** 35Q35, 35Q79.

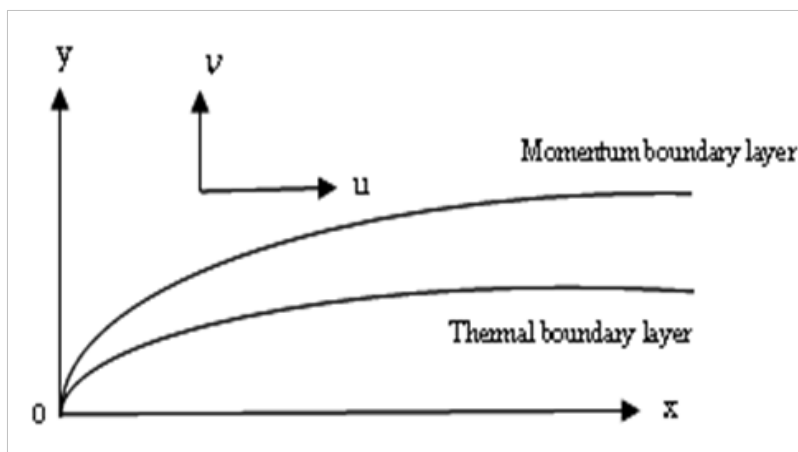
### 1. INTRODUCTION

The flow of fluid with suspended particulate matter (SPM) has received the attention of many researchers due to its possible industrial applications like sedimentation, pipe flows, fluidized beds, gas-purification and transport process. Soo [1] and Chiu [2] have studied the boundary layer flow of fluid with suspended particles by neglecting the particle momentum equation in the normal direction. Both Marble [3] and Soo [7] have developed the conservation laws of mass, momentum and energy for two-phase flow and have obtained their closed form solutions. Marble [3] has considered dynamics of a gas combining small solid particles and obtained the solutions valid for far downstream region of the plate assuming zero particulate velocity on the surface. Soo [7] has derived momentum integrals for the fluid and particle phases. Soo [4] has investigated the laminar mixing of a suspension with a clean fluid without considering the conservation of particulate phase momentum in the normal direction. Rudinger [5] has shown the effect of finite particle volume on the dynamics of gas-particle mixtures. Singleton [6] has considered compressible laminar boundary-layer

flow of a dusty gas over a semi-infinite flat plate and obtained asymptotic solutions using the series expansion method for both small slip region and the large slip region close to the leading edge of the plate. Otterman [8] has studied the laminar mixing of a dusty fluid with clean fluid and has shown the effect of transverse force on the flow field. Tabakoff and Hamed [9] have studied boundary layer flow of particulate gas and pointed out that particle velocity decreases linearly and particle density increases continuously along the plate length. Jain and Ghosh [10] has studied the laminar boundary layer flow on a flat plate employing momentum integral method. Prabha and Jain [11] have employed the finite difference technique to study the laminar boundary layer flow over a flat plate, but have not considered the momentum equation in the normal direction. Dutta and Mishra [12] have studied Boundary Layer Flow of a Dusty Fluid over a Semi-Infinite Flat Plate. Wang and Glass [13] have studied compressible laminar boundary layer flows of a dusty gas over a semi-infinite flat plate considering a moderate slip region (a non-equilibrium transition region) in addition to the large and small slip regions. Chamakha [14] has considered a continuum mathematical model governing compressible boundary-layer fluid-particle flow and heat transfer over a semi-infinite flat plate. Panda et.al. [15] have studied the effect of diffusion of particles on the laminar two phase flow. Partha et. al. [16] have presented a similarity solution for mixed convection flow and heat transfer from an exponentially stretching surface by considering viscous dissipation effect in the medium. They showed that the buoyancy and viscous dissipation have significant influence on the non-dimensional skin friction and heat transfer coefficient. Panda et.al. [17] have studied the effect of volume- fraction and diffusion of SPM in in free convection flows in the vicinity of heated horizontal flat plate. Mishra and Tripathy [18, 20] have investigated the two-phase boundary layer flow over a flat plate to study the boundary layer flow characteristics by using momentum integral method, where effect of volume fraction on the flow has not been studied. Misra et.al.[19] have employed the Crank-Nicholson finite difference technique to show the effect of electrification of suspended particulate matter(SPM) in a two phase boundary layer flow and heat transfer over a semi-infinite flat plate. From most of the above literature, it is observed that studies relating to the boundary layer flow of dusty fluid with negligible volume fraction, omission of the momentum equation for particulate phase in normal direction and assumption of no slip condition for particle velocity are inadequate. The assumption of neglecting the volume fractions of the particles is not justified when the fluid density is high or particle mass fraction is large. Rudinger [5] has shown that the error in neglecting the volume fraction ranges from insignificant to large. Otterman [8] has shown that the boundary layer approximation of the momentum equation for the fluid phase is not necessary and that the particle momentum equation in the normal direction cannot be neglected. Further, the assumption of no

slip condition for particle velocity in most of the above literature is not physically plausible since the particles do not flow in unison with fluid. In the boundary layer, the fluid decelerates from its free stream velocity to zero velocity at the solid surface, but since the density of the SPM is much greater than the fluid density, the SPM cannot accommodate this rapid deceleration but tends to slip through the fluid as they decelerate.

In the present problem, the influencing parameters like finite volume fraction, the momentum equation for particulate phase in the direction normal to the flow, heat due to conduction and viscous dissipation have been simultaneously considered to study their effects on the boundary layer flow and heat transfer of a steady viscous incompressible fluid with uniformly distributed suspended particles past a thin heated semi-infinite flat plate fulfilling the inadequacies of previous investigators. The diffusion equation is considered for calculating the particle concentration in the incompressible flow field. The heat due to conduction and viscous dissipation are included in the energy equations of both phases with a view to estimate the temperature profile as well as the rate of heat transfer on the surface of both phases. Further, the "no slip" condition is not satisfied by the particles, so the velocity, temperature and concentration of particle phase on the plate are considered as the functions of  $x$  only. The governing coupled, non-linear partial differential equations of the flow field are solved numerically using the finite difference technique with non-uniform-grid.



**Figure 1.** Schematic diagram of the flow field

## NOMENCLATURE

$(u, v)$	Velocity components for the fluid phase in x- and y- directions respectively	$\mu$	Coefficient of viscosity of fluid
$(u_p, v_p)$	Velocity components for the particle phase in x-and y- directions respectively	$\delta$	Boundary layer thickness
$(T, T_p)$	Temperatures of fluid and particle phase	$\tau_T$	Thermal equilibrium time
$(T_w, T_\infty)$	Temperature at the wall and free-stream respectively	$T_0$	Temperature of the plate at $\eta=0$
$(\nu, \nu_p)$	Kinematics coefficient of viscosity of fluid and particle phase respectively	$a$	Thermal diffusivity
$(\rho, \rho_p)$	Density of fluid and particle phase respectively	$\kappa$	Thermal conductivity
$(\rho_s, \rho_m)$	Material density of particle and mixture respectively	$\alpha$	Concentration parameter
$P_r$	Prandtl number	$\epsilon$	Diffusion parameter
$E_c$	Eckret number	$J_{max}$	Maximum number of Grid points along Y-axis
$N_u$	Nusselt number	$L$	Reference length
$C_f$	Skin friction coefficient	$W(x, y)$	Dummy variable
$\varphi$	Volume fraction of SPM	$r_y$	Grid growth ratio
$D$	Diameter of the particle	$D_p$	Binary diffusion coefficient
$U$	Free stream velocity		

## 2. MATHEMATICAL FORMULATION

A steady flow of a viscous incompressible fluid with uniformly distributed suspended particles past a heated thin semi-infinite flat plate at a constant temperature  $T_w$  is considered. The plate is placed along the direction of a uniform free stream of velocity  $U$  and temperature  $T_\infty$ . Let the plate be placed in the direction of X-axis and Y-axis be perpendicular to it, as shown in Fig.1. Introducing the non-dimensional variables

$$\begin{aligned}
 x^* &= x/L, & n &= y/LRe \\
 u^* &= u/U, & v^* &= v/URe \\
 u_p^* &= u_p/U, & v_p^* &= v_p/URe \\
 (2.1) \quad T^* &= \frac{T - T_\infty}{T_w - T_\infty}, & T_p^* &= \frac{T_p - T_\infty}{T_w - T_\infty} & \rho_p^* &= \frac{\rho_p}{\rho_{p0}}
 \end{aligned}$$

The governing boundary layer equations of the flow field following Misra et. al. [19], after dropping stars are given by

$$(2.2) \quad \frac{\partial u}{\partial x} + \frac{\partial v}{\partial \eta} = 0$$

$$(2.3) \quad \frac{\partial \rho_p}{\partial x} + v_p \frac{\partial \rho_p}{\partial \eta} = \epsilon \frac{\partial^2 \rho_p}{\partial \eta^2}$$

$$(2.4) \quad u \frac{\partial u}{\partial x} + \nu \frac{\partial u}{\partial \eta} = \frac{\partial^2 u}{\partial \eta^2} - \alpha \frac{\psi}{1-\psi} \frac{FL}{U} \rho_p (u - u_p)$$

$$(2.5) \quad u_p \frac{\partial u_p}{\partial x} + v_p \frac{\partial u_p}{\partial \eta} = \epsilon \frac{\partial^2 u_p}{\partial \eta^2} + \frac{FL}{U} (u - u_p)$$

$$(2.6) \quad u_p \frac{\partial v_p}{\partial x} + v_p \frac{\partial v_p}{\partial \eta} = \epsilon \frac{\partial^2 v_p}{\partial \eta^2} + \frac{FL}{U} (v - v_p)$$

$$(2.7) \quad u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial \eta} = \frac{1}{Pr} \frac{\partial^2 T}{\partial \eta^2} + Ec \left( \frac{\partial u}{\partial \eta} \right)^2 + \frac{2}{3} \frac{\alpha}{Pr} \frac{\psi}{1-\psi} \frac{FL}{U} \rho_p (T_p - T)$$

$$(2.8) \quad u_p \frac{\partial T_p}{\partial x} + v_p \frac{\partial T_p}{\partial \eta} = \frac{FL}{U} (T - T_p) + \frac{\epsilon}{Pr} \frac{\partial^2 T_p}{\partial \eta^2} + \epsilon \cdot Ec \left[ \left( \frac{\partial u_p}{\partial \eta} \right)^2 + u_p + p \frac{\partial^2 u_p}{\partial \eta^2} \right]$$

$$(2.9) \quad \eta = 0 : u = 0, v = 0, u + p = u_{pw}(x), v_p = 0, \rho_p = \rho_{pw}(x), T = 1, T_p = T_{pw}(x)$$

$$(2.10) \quad \eta = \infty : u = u_p = \rho_p = 1, v_p = 0, T = 0, T_p = 0$$

Introducing the finite difference expressions with non-uniform-grid for the various terms. The Equations (2.2) to (2.8) are reduced to

$$v_j^{n+1} = v_{j-1}^{n+1} - \frac{1}{2} \frac{\Delta y}{\Delta x} \left[ (1.5u_j^{n+1} - 2u_j^n + 0.5u_j^{n-1}) + (1.5u_{j-1}^{n+1} - 2u_{j-1}^n + 0.5u_{j-1}^{n-1}) \right]$$

$$a_j u_{j-1}^{n+1} + b_j u_j^{n+1} + c_j u_{j+1}^{n+1} = d_j$$

$$a_j^* u_{p_{j-1}}^{n+1} + b_j^* + u_{p_j}^{n+1} + c_j^* u_{p_{j+1}}^{n+1} = d_j^*$$

$$a_j^{**} v_{p_{j-1}}^{n+1} + b_j^{**} v_{p_j}^{n+1} + c_j^{**} v_{p_{j+1}}^{n+1} = d_j^{**}$$

$$a_j^+ T_{j-1}^{n+1} + b_j^+ T_j^{n+1} + c_j^+ T_{j+1}^{n+1} = d_j^+$$

$$a_j^+ T_{p_{j-1}}^{n+1} + b_j^{++} T_{p_j}^{n+1} = d_j^{++}$$

$$a_j^\# \rho_{p_{j-1}}^{n+1} + b_j^\# \rho_{p_j}^{n+1} + c_j^\# \rho_{p_{j+1}}^{n+1} = d_j^\#$$

Where,

$$\begin{aligned}
a_j &= \frac{1}{\Delta x} [-pr_y - q] \\
b_j &= \frac{1}{\Delta x} \left[ 1.5(2u_j^n - u_j^{n-1}) + p \left( r_y - \frac{1}{r_y} \right) + q \left( 1 + \frac{1}{r_y} \right) + \frac{\psi}{1-\psi} \frac{FL}{U} \alpha \Delta x \left( 2\rho_{pj}^n - \rho_{pj}^{n-1} \right) \right] \\
c_j &= \frac{1}{\Delta x} \left[ \frac{1}{r_y} (p - q) \right] \\
d_j &= \frac{1}{\Delta x} \left[ (2u_j^n - u_j^{n-1})(2u_j^n - 0.5u_j^{n-1}) - \frac{\psi}{1-\psi} \frac{FL}{U} \alpha \Delta x \left( 2\rho_{pj}^n - \rho_{pj}^{n-1} \right) (-2u_{pj}^n + u_{pj}^{n-1}) \right] \\
a_j^* &= \frac{1}{\Delta x} [-pr_y - \epsilon q] \\
b_j^* &= \frac{1}{\Delta x} \left[ 1.5(2u_{pj}^n - u_{pj}^{n-1}) + p \left( r_y - \frac{1}{r_y} \right) + \epsilon q \left( 1 + \frac{1}{r_y} \right) + \frac{FL}{U} \Delta x \right] \\
c_j^* &= \frac{1}{\Delta x} \left[ \frac{1}{r_y} (p - \epsilon q) \right] \\
d_j^* &= \frac{1}{\Delta x} \left[ (2u_{pj}^n - u_{pj}^{n-1})(2u_{pj}^n - 0.5u_{pj}^{n-1}) + \frac{FL}{U} \Delta x u_j^{n+1} \right] \\
a_j^{**} &= \frac{1}{\Delta x} [-pr_y - \epsilon q] \\
b_j^{**} &= \frac{1}{\Delta x} \left[ 1.5u_{pj}^{n+1} + p \left( r_y - \frac{1}{r_y} \right) + \epsilon q \left( 1 + \frac{1}{r_y} \right) + \frac{FL}{U} \Delta x \right] \\
c_j^{**} &= \frac{1}{\Delta x} \left[ \frac{1}{r_y} (p - \epsilon q) \right] \\
d_j^{**} &= \frac{1}{\Delta x} \left[ u_{pj}^{n+1} (2v_{pj}^n - 0.5v_{pj}^{n-1}) + \frac{FL}{U} \Delta x v_j^{n+1} \right] \\
a_j^+ &= \frac{1}{\Delta x} \left[ -q \left( 0.5r_y \Delta y v_j^{n+1} + \frac{1}{Pr} \right) \right] \\
b_j^+ &= \frac{1}{\Delta x} \left[ 1.5u_j^{n+1} + 0.5q \Delta y v_j^{n+1} \left( r_y - \frac{1}{r_y} \right) + \frac{1(1+r_y)}{Pr \cdot r_y} + \frac{2\alpha}{2Pr} \frac{\psi}{1-\psi} \frac{FL}{U} \Delta x \rho_{pj}^{n+1} \right] \\
c_j^+ &= \frac{1}{\Delta x} \left[ \frac{q}{r_y} \left( 0.5 \Delta y v_j^{n+1} - \frac{1}{Pr} \right) \right] \\
d_j^+ &= \frac{1}{\Delta x} \left[ \frac{2\alpha}{3Pr} \frac{\psi}{1-\psi} \frac{FL}{U} \rho_{pj}^{n+1} (2T_{pj}^n - T_{pj}^{n-1}) \Delta x + \Delta \right] + \\
&\quad \frac{1}{\Delta x} \left[ x \cdot Ec \left( \frac{u_{j+1}^{n+1} - u_j^{n+1}}{\Delta y} \right)^2 + u_j^{n+1} (2T_j^n - 0.5T_j^{n-1}) \right]
\end{aligned}$$

$$\begin{aligned}
 a^{++} &= \frac{1}{\Delta x} \left[ -q \left( 0.5r_y \Delta y v_j^{n+1} + \frac{\epsilon}{\text{Pr}} \right) \right] \\
 b_j^{++} &= \frac{1}{\Delta x} \left[ 1.5u_{p_j}^{n+1} + 0.5q \Delta y v_{p_j}^{n+1} \left( r_y - \frac{1}{r_y} \right) + \frac{\epsilon q (1 + r_y)}{\text{Pr} \cdot r_y} + \frac{FL}{U} \Delta x \right] \\
 c_j^{++} &= \frac{1}{\Delta x} \left[ \frac{q}{r_y} \left( 0.5 \Delta y \cdot v_{p_j}^{n+1} - \frac{\epsilon}{\text{Pr}} \right) \right] \\
 d^{++} &= \frac{1}{\Delta x} \left[ u_{p_j}^{n+1} (2T_{p_j}^n - 0.5T_{p_j}^{n-1}) + \frac{FL}{U} T_j^{n+1} \Delta x \right] + \\
 \epsilon \cdot Ec &\left[ \left( \frac{u_{p_{j+1}}^{n+1} - u_{p_j}^{n+1}}{\Delta y} \right)^2 + u_{p_j}^{n+1} \left( \frac{u_{p_{j-1}}^{n+1} - \left( 1 + \frac{1}{r_y} \right) u_{p_j}^{n+1} + \frac{1}{r_y} u_{p_{j+1}}^{n+1}}{(1 + r_y) \Delta y^2} \right) \right] \\
 a^\# &= -v_{p_j}^{n+1} r_{y_i}^2 \Delta y - 2\epsilon r_y \\
 b_j^\# &= \frac{1.5p^\# u_{p_j}^{n+1}}{\Delta x} - v_{p_j}^{n+1} (1 - r_y)^2 \Delta y + 2\epsilon (1 + r_y) \\
 c_j^\# &= v_{p_j}^{n+1} \Delta y - 2\epsilon \\
 d_j^\# &= p^\# u_{p_j}^{n+1} \frac{2\rho_{p_j}^n - 0.5\rho_{p_j}^{n-1}}{\Delta x} \\
 p &= (2v_j^n - v_j^{n-1}) \frac{\Delta \Delta}{(1 + r_y) \Delta y} \\
 q &= \frac{2\Delta x}{(1 + r_y) \Delta y^2} \quad p^\# = r_y (1 + r_y) \Delta y
 \end{aligned}$$

Using the boundary conditions (2.9) and (2.10) at  $j = 1$  or  $j = j_{\max}$ , we get

$$\begin{aligned}
 a_2 &= 0, \quad u_1 = 0 \quad \text{at } j = 2 \\
 d_j &= d_j - c_j u_e \quad \text{at } j = j_{\max} - 1 \\
 d_2^* &= d_2^* - a_2^* v_{pw} \quad \text{at } j = 2 \\
 d_j^* &= d_j^* - c_j u_e \quad \text{at } j = j_{\max} - 1 \\
 a_2^{**} &= 0 \quad \text{at } j = 2 \\
 c_j^{**} &= 0 \quad \text{at } j = j_{\max} - 1 \\
 d_2^+ &= d_2^+ - a_2^+ \quad \text{at } j = 2 \\
 d_2^+ &= d_2^+ - a_2^+ \quad \text{at } j = j_{\max} - 1 \\
 c_j^+ &= 0 \quad \text{at } j = j_{\max} - 1 \\
 d_2^{++} &= d_2^{++} - a_2^{++} T_{pw} \quad \text{at } j = 2 \\
 c_j^{++} &= 0 \quad \text{at } j = j_{\max} - 1 \\
 d_2^\# &= d_2^\# - a_2^\# \rho_{pw} \quad j = 2 \\
 d_j^\# &= d_j^\# - c_j^\# \quad \text{at } j_{\max} - 1
 \end{aligned}$$

As no slip condition is not satisfied by the particles, so  $u_{pw}$ ,  $\rho_{pw}$ , and  $T_{pw}$  are calculated separately on the plate at  $\eta = 0$ . As  $u_{pw}$ ,  $\rho_{pw}$ , and  $T_{pw}$  are functions of  $x$  only, so from Eqns. (2.5), (2.3) and (2.8) we obtain

$$(2.11) \quad \frac{\partial u_{pw}}{\partial x} = -\frac{FL}{U}$$

$$(2.12) \quad \frac{\partial}{\partial x}(\rho_{pw}u_{pw}) + \frac{\partial}{\partial y}(\rho_p v_p) = 0 \Rightarrow u_{pw} \frac{\partial \rho_{pw}}{\partial x} - \rho_{pw} \frac{FL}{U} = 0$$

$$(2.13) \quad u_{pw} \frac{\partial T_{pw}}{\partial x} = \frac{FL}{U}(1 - T_{pw})$$

Using finite differences, Eqns. (2.11), (2.12), and (2.13) are reduced to

$$(2.14) \quad u_1^{n+1} = -\frac{2FL}{3U}\Delta x + \frac{4}{3}u_1^n - \frac{1}{3}u_1^{n-1}$$

$$(2.15) \quad \rho_{p1}^{n+1} = \frac{2\rho_{p1}^n - 0.5\rho_{p1}^{n-1}}{1.5 - \frac{FL\Delta x}{U u_{p1}^{n+1}}}$$

$$(2.16) \quad T_{p1}^{n+1} = \frac{2T_{p1}^n - 0.5T_{p1}^{n-1} + \frac{FL}{U} \frac{\Delta x}{u_{p1}^{n+1}}}{1.5 + \frac{FL}{U} \frac{\Delta x}{u_{p1}^{n+1}}}$$

### 3. Heat Transfer

The heat transfer characteristic is expressed in terms of Nusselt number, given by

$$(3.1) \quad Nu^{n+1} = -\sqrt{Re} \left[ \frac{\partial T}{\partial \eta} \right]_{\eta=0}^{n+1} = -\sqrt{Re} \left[ \frac{T_{j+1}^{n+1} - (1 - r_y^2)T_j^{n+1} - r_y^2 T_{j-1}^{n+1}}{r_y(1 + r_y)\Delta y} \right]_{j=2}$$

$$= -\sqrt{Re} \left[ \frac{T_3^{n+1} - (1 - r_y^2)T_2^{n+1} - r_y^2 T_1^{n+1}}{r_y(1 + r_y)\Delta y} \right]$$

### 4. RESULTS AND DISCUSSION

The values of the various parameters involved are chosen as

$$\rho = 0.9752 \text{ kg/m}^3 \quad \rho_p = 800, 2403, 8010 \text{ kg/m}^3 \quad \epsilon = 0.05, 0.1, 0.2$$

$$d = 50, 100 \quad \mu_m U = 60.96 \text{ msec} \quad \mu = 1.5415 \times 10^{-5} \text{ kg(m sec)}$$

$$\varphi = 0.001, 0.0003, 0.0001 \quad L = 0.3048 \text{ m} \quad \alpha = 0.1 \text{ Ec} = 0.1 \quad Pr = 0.71, 1.0, 7.0$$

The accuracy of the numerical scheme is validated by comparing the results of Nusselt number and Displacement thickness obtained from the present investigation with the results obtained by Panda et.al.[17] and Mishra and Tripathy [20] for negligible volume fraction  $\psi$  and omission of the momentum equation  $v_p$  for particulate phase in normal direction. It is observed from the Tables 1 and 2 that the present result is in good



agreement with the results obtained by Panda et.al.[17] and Mishra and Tripathy [20] for  $\rho_p = 800, \epsilon = 0.05, Pr = 0.71, d = 50, \varphi = 0.0, \alpha = 0.1$  and  $Ec = 0.1$ .

Again the present results of Nusselt number and Displacement thickness for  $\varphi=0.0$  without  $v_p$  are compared with that of obtained for  $\varphi=0.0001$  with  $v_p$ . It is envisaged from the 3rd and 4th columns of Tables 1 and 2 that the rate of heat transfer in terms of Nusselt number as well as the Displacement thickness increase in case of  $\varphi=0.0001$  with  $v_p$  as compared to that of  $\varphi=0.0$  without  $v_p$  along the plate.

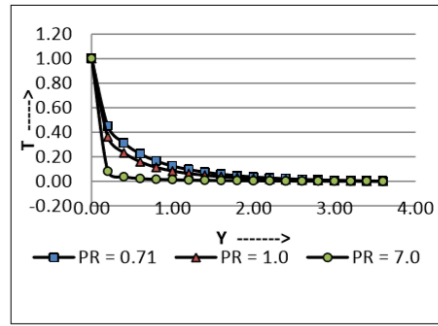
**Table 1.** Comparison of Nusselt number Nu for Particulate suspension .

$X$	Panda et.al.[17] Nu with $\varphi = 0.0$ and without $v_p$	Mishra [20] Nu with $\varphi = 0.0$ and without $v_p$ $\mu$	Present Study Nu with $\varphi = 0.0$ and without $v_p$	Present Study Nu with $\varphi = 0.0001$ and with $v_p$
1.20	4.54E+03	4.59E+03	4.57E+03	6.29E+02
2.00	4.37E+03	4.36E+03	4.38E+03	5.10E+03
2.80	3.37E+03	3.37E+03	3.37E+03	3.43E+03
3.60	3.36E+03	3.35E+03	3.37E+03	3.44E+03
4.40	3.36E+03	3.34E+03	3.36E+03	3.44E+03
5.00	3.37E+03	3.34E+03	3.37E+03	3.44E+03

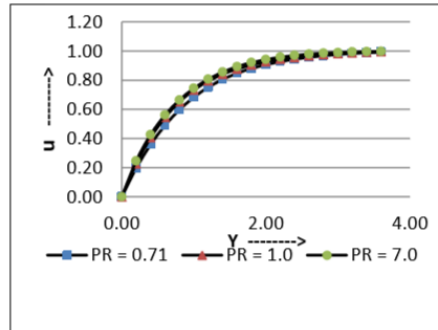
**Table 2.** Comparison of Displacement thickness (DISP) for Particulate suspension.

$X$	Panda et.al.[17] DISP with $\varphi = 0.0$ and without $v_p$	Mishra [20] DISP with $\varphi = 0.0$ and without $v_p$ $\mu$	Present Study DISP with $\varphi = 0.0$ and without $v_p$	Present Study DISP with $\varphi = 0.0001$ and with $v_p$
1.20	5.37E-03	5.39E-03	5.34E-03	4.76E-03
2.00	4.66E-03	4.63E-03	4.64E-03	5.09E-03
2.80	4.35E-03	4.38E-03	4.36E-03	7.84E-03
3.60	4.27E-03	4.26E-03	4.28E-03	7.81E-03
4.40	4.24E-03	4.24E-03	4.25E-03	7.81E-03
5.00	4.24E-03	4.24E-03	4.25E-03	7.81E-03

To study the effect of various physical parameters on the velocity field, thermal boundary layer and coefficient of rate of heat transfer on the wall, the results obtained from numerical computation is depicted through Fig.2 to 10 and Tables 3 to 9. The values of Prandtl number Pr are taken as 0.71, 1.0 and 7.0 which physically correspond to air, electrolyte solution and water, respectively.



**Figure 2.** Variation of normalized temperature of carrier fluid  $T$  for different values of  $Pr$

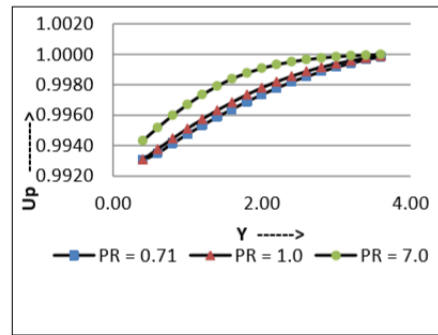


**Figure 3.** Variation of non-dimensional velocity profile of carrier fluid  $u$  for different values of  $Pr$

**Table 3.** Variation of normalized Particle temperature  $T_p$  for different values of Prandtl number  $Pr$ .

$Y$	$T_p$ for $Pr = 0.71$	$T_p$ for $Pr = 1.0$	$T_p$ for $Pr = 7.0$
0.00	2.63E-02	2.63E-02	2.63E-02
0.40	1.03E-03	7.03E-04	2.85E-05
0.80	6.23E-04	3.98E-04	7.15E-07
1.20	3.79E-04	2.33E-04	1.68E-06
1.60	2.31E-04	1.37E-04	1.74E-06
2.00	1.40E-04	8.08E-05	1.15E-06
2.40	8.23E-05	4.62E-05	6.34E-07
2.80	4.53E-05	2.48E-05	3.11E-07
3.20	2.14E-05	1.14E-05	1.29E-07
3.60	5.70E-06	2.97E-06	3.05E-08

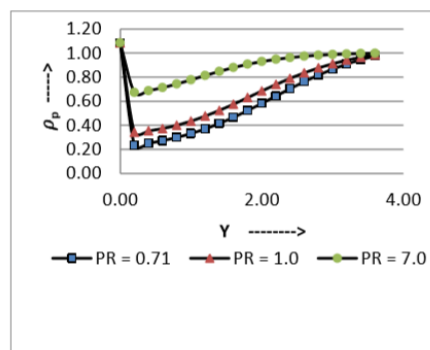
From Fig.2, it is observed that the normalized temperature of carrier fluid  $T$  is higher for air ( $Pr = 0.71$ ) as well as the temperature distribution is more uniform across the thermal boundary layer as compared to water ( $Pr=7.0$ ) and electrolyte solution ( $Pr=1.0$ ). Further it is concluded that heat is diffused away from the heated



**Figure 4.** Variation of non-dimensional particle velocity up for different values of Pr

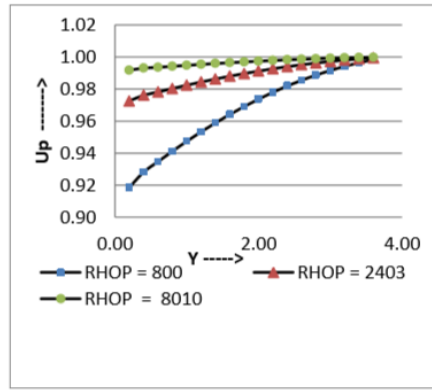
**Table 4.** Variation of non-dimensional particle density  $\rho_p$  for different values of material density of particles  $\rho_s$

Y	$\rho_s = 800$	$\rho_s = 2403$	$\rho_s = 8010$
0.00	1.3462	1.1409	1.0834
0.40	0.2477	0.2490	0.2499
0.80	0.2974	0.2979	0.2985
1.20	0.3683	0.3688	0.3693
1.60	0.4644	0.4652	0.4657
2.00	0.5804	0.5813	0.5818
2.40	0.7030	0.7038	0.7041
2.80	0.8162	0.8168	0.8170
3.20	0.9084	0.9087	0.9088
3.60	0.9754	0.9755	0.9755

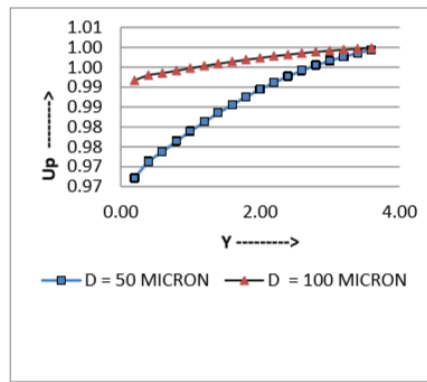


**Figure 5.** Variation of non-dimensional particle density  $\rho_p$  or different values of Pr

surface more rapidly for higher values of Prandtl number Pr. The normalized temperature is maximum near the plate and it asymptotically approaches to the free stream value towards the boundary layer. The same trend is noticed in case of normalized particle temperature  $T_p$  from Table 3.



**Figure 6.** Variation of non-dimensional particle velocity up for different values of material density of particles  $\rho_s$

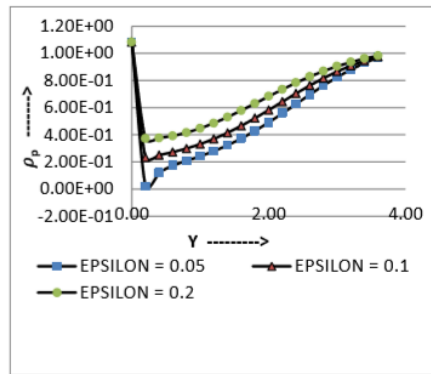


**Figure 7.** Variation of non-dimensional particle velocity up for different size of particles D

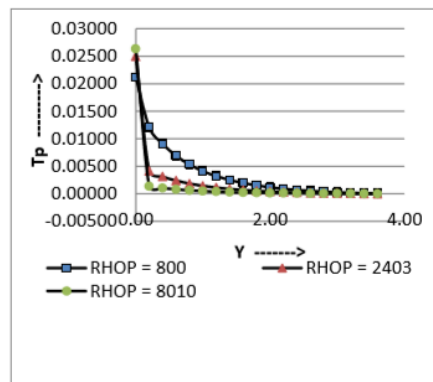
**Table 5.** Variation of non-dimensional Particle phase density  $\rho_p$  for different size of particles D.

Y	$\rho_p$ for D= 50 mi- cron	$\rho_p$ for D= 100 mi- cron
0.00	1.1585	1.0834
0.40	0.2488	0.2499
0.80	0.2978	0.2985
1.20	0.3687	0.3693
1.60	0.4651	0.4657
2.00	0.5812	0.5818
2.40	0.7037	0.7041
2.80	0.8167	0.8170
3.20	0.9087	0.9088
3.60	0.9755	0.9755

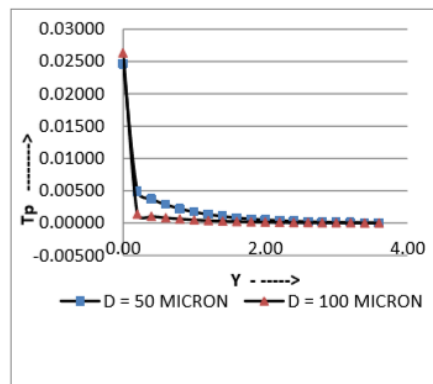
From Fig. 3, 4 and 5, it is observed that the magnitudes of non-dimensional carrier fluid velocity up, non-dimensional particle velocity up, non-dimensional particle



**Figure 8.** Variation of non-dimensional particle density  $\rho_p$  for different values of  $\epsilon$



**Figure 9.** Variation of normalized particle temperature  $T_p$  for different values of material density of particles  $\rho_s$



**Figure 10.** Variation of normalized particle temperature  $T_p$  for different size of particles  $D$

phase density  $\rho_p$  increase with the increase of Prandtl number  $Pr$  for a fixed value of volume fraction  $\psi$  of the particle phase across the boundary layer.

The non-dimensional particle velocity up drastically increases near the plate attaining the free stream value towards the boundary layer due to the high material

**Table 6.** Variation of non-dimensional particle velocity up for various diffusion parameters  $\epsilon$ .

$Y$	up for $\epsilon = 0.05$	up for $\epsilon = 0.1$	up for $\epsilon = 0.2$
<i>0.00</i>	0.9217	0.9217	0.9217
<i>0.40</i>	0.9922	0.9930	0.9927
<i>0.80</i>	0.9939	0.9941	0.9942
<i>1.20</i>	0.9951	0.9953	0.9955
<i>1.60</i>	0.9962	0.9964	0.9966
<i>2.00</i>	0.9971	0.9974	0.9976
<i>2.40</i>	0.9980	0.9982	0.9984
<i>2.80</i>	0.9987	0.9989	0.9990
<i>3.20</i>	0.9993	0.9994	0.9995
<i>3.60</i>	0.9998	0.9998	0.9999

**Table 7.** Variation of normalized Particle temperature  $T_p$  for various diffusion parameters  $\epsilon$ 

$Y$	$T_p$ for $\epsilon = 0.05$	$T_p$ for $\epsilon = 0.1$	$T_p$ for $\epsilon = 0.2$
<i>0.00</i>	2.63E-02	2.63E-02	2.63E-02
<i>0.40</i>	7.31E-04	1.03E-03	1.24E-03
<i>0.80</i>	5.06E-04	6.23E-04	7.42E-04
<i>1.20</i>	3.07E-04	3.79E-04	4.56E-04
<i>1.60</i>	1.85E-04	2.31E-04	2.83E-04
<i>2.00</i>	1.10E-04	1.40E-04	1.73E-04
<i>2.40</i>	6.35E-05	8.23E-05	1.03E-04
<i>2.80</i>	3.44E-05	4.53E-05	5.73E-05
<i>3.20</i>	1.60E-05	2.14E-05	2.72E-05
<i>3.60</i>	4.22E-06	5.70E-06	7.31E-06

density of particles (Fig. 6), presence of coarser particles (Fig. 7) as well as high diffusion of particles through the carrier fluid (Table 6). The same trend is noticed in case of non-dimensional particle phase density  $\rho_p$  with respect to  $\rho_s$ ,  $D$  and  $\epsilon$  from Table 4, Table 5 and Fig. 8 respectively.

The normalized particle temperature  $T_p$  is reduced for higher material density of particles  $\rho_s$  (Fig. 9), where as it is enhanced for higher diffusion parameter  $\epsilon$  across the boundary layer (Table 7). From Fig.10, it is concluded that the normalized particle phase temperature  $T_p$  decreases across the boundary layer with the increase of the size of particles. Corresponding to the carrier fluid with SPM, it is observed that the Nusselt number  $Nu$  increases (Table 8) and the displacement thickness decreases (Table 9) for the higher values of the Prandtl number  $Pr$  along the plate through the flow field.

**Table 8.** Variation of Nusselt number Nu for different values of Prandtl number Pr

Y	Nu for clear fluid	Nu for fluid with SPM where $\psi = 0.0001$	Nu for fluid with SPM with $\psi=0.0001, Pr = 0.71$	Nu for fluid with SPM with $\psi=0.0001, Pr = 1.0$	Nu for fluid with SPM with $\psi=0.0001, Pr = 7.0$
1.20	2.63E+03	6.29E+02	8.52E+01	3.16E+02	5.12E+03
1.60	3.31E+03	4.72E+03	1.20E+04	2.08E+04	1.51E+05
2.00	3.31E+03	5.10E+03	1.09E+04	1.94E+04	1.34E+05
2.40	3.31E+03	2.06E+03	1.17E+04	2.01E+04	1.32E+05
2.80	3.31E+03	3.43E+03	1.11E+04	1.98E+04	1.33E+05
3.20	3.31E+03	3.44E+03	1.15E+04	1.98E+04	1.32E+05
3.60	3.31E+03	3.44E+03	1.13E+04	2.00E+04	1.32E+05
4.00	3.31E+03	3.44E+03	1.14E+04	1.97E+04	1.32E+05
4.40	3.31E+03	3.44E+03	1.13E+04	1.99E+04	1.32E+05
4.80	3.31E+03	3.44E+03	1.14E+04	1.98E+04	1.32E+05
5.00	3.31E+03	3.44E+03	1.13E+04	1.98E+04	1.32E+05

**Table 9.** Variation of Displacement thickness (DISP) for different values of Prandtl number Pr

Y	DISP for clear fluid	DISP for fluid with SPM where=0.0001	DISP for fluid with SPM with $\psi=0.0001, Pr = 0.71$	DISP for fluid with SPM with $\psi=0.0001, Pr = 1.0$	DISP for fluid with SPM with $\psi=0.0001, Pr = 7.0$
1.20	5.37E-03	4.76E-03	4.24E-03	3.89E-03	2.87E-03
1.60	5.05E-03	3.82E-03	3.06E-03	2.77E-03	2.32E-03
2.00	4.66E-03	5.09E-03	3.68E-03	3.29E-03	2.11E-03
2.40	4.45E-03	7.75E-03	3.27E-03	3.04E-03	2.49E-03
2.80	4.35E-03	7.84E-03	3.55E-03	3.13E-03	2.31E-03
3.20	4.29E-03	7.82E-03	3.38E-03	3.14E-03	2.27E-03
3.60	4.27E-03	7.81E-03	3.49E-03	3.09E-03	2.41E-03
4.00	4.25E-03	7.81E-03	3.42E-03	3.16E-03	2.32E-03
4.40	4.24E-03	7.81E-03	3.46E-03	3.09E-03	2.32E-03
4.80	4.24E-03	7.81E-03	3.44E-03	3.15E-03	2.36E-03
5.00	4.24E-03	7.81E-03	3.46E-03	3.12E-03	2.33E-03

Further both the Nusselt number Nu and the displacement thickness corresponding to the carrier fluid with SPM of negligible volume fraction are more as compared to those respective values corresponding to the clear fluid throughout the flow field (Tables 8 and 9).

## 5. CONCLUSION

The main results of this investigation are briefly summarized as follows.

1. Higher Prandtl number is to increase the non-dimensional carrier fluid velocity, particle velocity and particle phase density across the boundary layer as well as to increase the Nusselt number along the plate throughout the flow field. The normalized temperatures of both carrier fluid and particle phase are reduced with the increase of Prandtl number envisaging that heat generated due to the flow of particulate suspension is diffused away from the heated surface more rapidly for higher values Prandtl number.
2. The non-dimensional particle velocity and particle phase density are increased with the increase of particle size, material density of particles and diffusion parameter across the boundary layer.

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